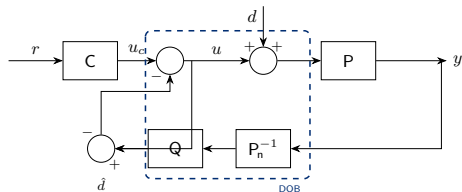


What a Classical DOB Does



- Estimate the “equivalent disturbance” via $\hat{d} = Q \cdot P_n^{-1}y - Q \cdot u$
- Subtract $\hat{d}$ from the control input
- Where $Q(j\omega) \approx 1$ (low frequencies), the output *approximately* recovers nominal behaviour:

$$y(j\omega) \approx \frac{P_n C}{1 + P_n C} r(j\omega)$$

Why Attenuation $\neq$ Rejection

The disturbance-to-output transfer function is

$$T_d(s) = \frac{P(1-Q)P_n}{P(Q+P_nC) + (1-Q)P_n}.$$

- Q is a *low-pass filter*.
 $Q(j\omega) \approx 1$ for small ω , but $Q \neq 1$ exactly.
- So $(1-Q) \neq 0 \implies T_d \neq 0$
- For a **persistent sinusoid** $d(t) = A \sin(\sigma t + \phi)$:
a nonzero $T_d(j\sigma)$ means a *nonzero steady-state residual* — forever.

*Placeholder:
 $|T_d(j\omega)|$ sketch
nonzero at $\omega = \sigma$*

Attenuation is not cancellation. Can we do better?

The Key Idea: Embed the Disturbance Model

Internal Model Principle (Francis & Wonham, 1976)

To reject a class of disturbances *exactly*, the controller must contain a copy of the dynamics that generate them.

Classical DOB

- Low-pass $Q \rightarrow$ no disturbance model
- Attenuates, never cancels

Internal-Model DOB (this work)

- Q_q *embeds* the disturbance's minimal polynomial
- Asymptotic rejection: $e(t) \rightarrow 0$

The formal framework is *output regulation*;
we borrow exactly one ingredient — the internal model — and wire it into a DOB.

Problem Setup: Plant & Disturbance

Uncertain SISO plant (mechanical positioning system):

$$P(s) = \frac{1}{Ms^2 + Bs}, \quad P_n(s) = \frac{1}{M_n s^2 + B_n s}$$

- Mass M and damping B are *uncertain*; nominal values M_n, B_n are known.
- Input gain $g \in [g_{\min}, g_{\max}]$ — bounded but unknown.

Disturbance generated by an *exosystem*:

$$\dot{w} = S w, \quad d = R w$$

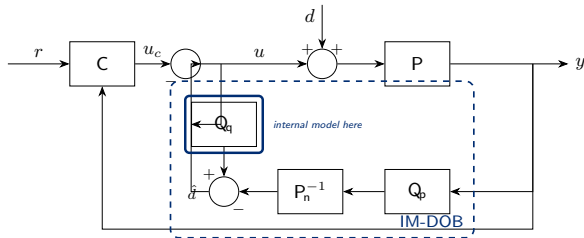
- The **minimal polynomial** $\Delta_w(s)$ of S is **known** (i.e. the disturbance *frequencies* are known).
- The amplitude and phase — encoded in R and $w(0)$ — are **unknown**.

Two Disturbance Scenarios

	Constant sinusoid	Growing sinusoid
Signal	$d(t) = A \sin(\sigma t + \phi)$	$d(t) = A t \sin(\sigma t + \phi)$
Minimal poly.	$\Delta_w(s) = s^2 + \sigma^2$	$\Delta_w(s) = (s^2 + \sigma^2)^2$
Order μ	2	4

- **Same framework**, same filter design procedure.
- The growing case $t \cdot \sin(\sigma t)$ is a disturbance that *no finite-gain feedback* (including PID) can reject — but the internal model can.

Architecture: The Internal-Model DOB



- Q_p : conventional low-pass (implements P_n^{-1} causally)
- Q_q : **internal-model filter** — encodes $\Delta_w(s)$
- $\hat{d} = Q_p P_n^{-1} y - Q_q u$ (same DOB topology, different Q)

Where the Internal Model Lives: the Q_q Filter

Define the *feedforward filter* $F(s) = \frac{1}{1 - Q_q(s)}$, realized as

$$\dot{\xi} = \Phi \xi + J v, \quad u = H \xi + v,$$

where Φ is the companion matrix of $\Delta_w(s)$.

Key fact

The closed-loop filter matrix $A_q + B_q C_q$ has characteristic polynomial

$$\det(sI - (A_q + B_q C_q)) = \Delta_w(s) = s^\mu + a_{\mu-1}s^{\mu-1} + \cdots + a_0.$$

*The Q_q filter **automatically** contains an internal model of the exosystem $\dot{w} = Sw$.*

Choosing the Filter: One Free Polynomial

$$Q_q(s) = \frac{\gamma_{\mu-1} s^{\mu-1} + \cdots + \gamma_0}{s^{\mu} + \alpha_{\mu-1} s^{\mu-1} + \cdots + \alpha_0}$$

Design recipe:

1. Pick *any* Hurwitz polynomial for the denominator:

$$s^{\mu} + \alpha_{\mu-1} s^{\mu-1} + \cdots + \alpha_0 \quad (\text{e.g. } (s+1)^{\mu}, \text{ Butterworth, } \dots)$$

2. The numerator coefficients are *determined*:

$$\gamma_i = \alpha_i - a_i, \quad i = 0, \dots, \mu-1$$

where a_i are the coefficients of $\Delta_w(s)$.

- No Vandermonde matrix to invert
(unlike prior work — those methods break when two disturbance frequencies coincide).
- No additional observer or filter blocks needed.

Main Result — In Words

*If the combined filter matrix $\mathcal{A}_f$ is Hurwitz
and the time-scale parameter τ is small enough,*

**the tracking error converges to zero
— exactly, not merely bounded —**

*for every admissible plant uncertainty
and every disturbance whose frequency is modelled.*

Main Result — Formal Statement

Theorem (Awai, Kim & Park, IFAC WC 2026)

Let:

- (A1) Plant parameters (ϕ, g) lie in a known compact set; minimal polynomial $\Delta_w(s)$ of S is known.
- (A2) Nominal closed-loop matrix $\mathcal{A}_s$ is Hurwitz.
- (A3) Combined fast-subsystem matrix $\mathcal{A}_f$ is Hurwitz.

Then $\exists \tau^* > 0$ such that for all $0 < \tau < \tau^*$:

$$\lim_{t \rightarrow \infty} e(t) = 0 \quad \text{and} \quad \lim_{t \rightarrow \infty} \|q(t) - \Sigma w(t)\| = 0$$

for all admissible (ϕ, g) , all $w(0)$, and all (R, S) consistent with Δ_w .

- $q(t) \rightarrow \Sigma w(t)$: the filter state *locks on* to the exosystem — this is the internal model at work.

The Lyapunov Certificate

Singular perturbation decomposition:

$$\underbrace{(x, \zeta)}_{\text{slow: plant + controller}} \quad \underbrace{(\hat{p}, \hat{q})}_{\text{fast: } Q_p + Q_q}$$

$$\dot{\chi} = \mathcal{A}_s \chi + \dots, \quad \tau \dot{\xi} = \mathcal{A}_f \xi + \dots$$

Composite Lyapunov function:

$$V(\chi, \xi) = \chi^\top \mathcal{P}_s \chi + \xi^\top \mathcal{P}_f \xi$$

where $\mathcal{A}_s^\top \mathcal{P}_s + \mathcal{P}_s \mathcal{A}_s = -I$ and $\mathcal{A}_f^\top \mathcal{P}_f + \mathcal{P}_f \mathcal{A}_f = -I$.

Guarantee

$$\text{For } 0 < \tau < \tau^*: \quad \dot{V} \leq -\frac{1}{2}\|\chi\|^2 - \frac{1}{2\tau}\|\xi\|^2$$

*The existence of this V is the stability guarantee.
No need to walk through the inequalities.*

The Design Knob: τ

- τ sets the **time-scale separation** between the controller and the DOB filters.
- Smaller τ
 - $\Rightarrow$ faster observer & disturbance estimator
 - $\Rightarrow$ better transient rejection
- But the Lyapunov guarantee requires

$$0 < \tau < \tau^*$$

- τ^* depends on plant uncertainty bounds and the filter polynomials (α_i, β_i) .

τ is a single engineering knob with a computable safety limit.

*Placeholder:
response overlay for
 $\tau = 0.1, 0.05, 0.01$*

FAQ: “Isn’t This Just Integral Action / PID?”

Integral action is the internal model of a *constant*:

$$\Delta_w(s) = s \quad \longleftrightarrow \quad d(t) = \text{const.}$$

This work generalises to *any* modelled frequency:

$$\Delta_w(s) = s^2 + \sigma^2 \quad \text{or} \quad (s^2 + \sigma^2)^2 \quad \text{or} \quad \dots$$

Method	Rejects
PID	constant offset
This	$\sin(\sigma t)$
This	$t \cdot \sin(\sigma t)$

- A PID controller *cannot* reject $d(t) = t \sin(\sigma t)$ — no finite integral gain suffices.
- The internal-model DOB handles it by embedding $(s^2 + \sigma^2)^2$.

FAQ: “What If the Frequency σ Is Unknown?”

Adaptive-DOB extension

- Replace the fixed coefficients a_i in $\Delta_w(s)$ with *online-estimated* coefficients $\hat{a}_i(t)$.
- The filter structure is identical; only the parameters adapt.
- **Status:** current work — stay tuned.

For today's results, σ is known.

The amplitude and phase remain unknown — and are handled.